

CFD SIMULATIONS OF POLLUTANT SPATIAL DISTRIBUTION IN A LARGE OFFICE

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Motivation

- **People spend most of their time indoors**
- **Indoors: many pollution sources
(for example: equipment and construction materials)**
- **Indoor CFD:**
 - **No best practice guidelines (in contrast to external flows)**
 - **Very few basic research cases
(usually specific problems are examined)**
 - **Not clear enough which physical and numerical parameters
affect the modelling results**
 - **Confinement of flow increases the possibilities of existence
of spots with unsteady flow phenomena**

Objectives

- ***Examine the flow/ dispersion in a large office***
- ***Focus on differences of concentrations among various working positions of employees***
- ***Examine influence of physical/modelling parameters***

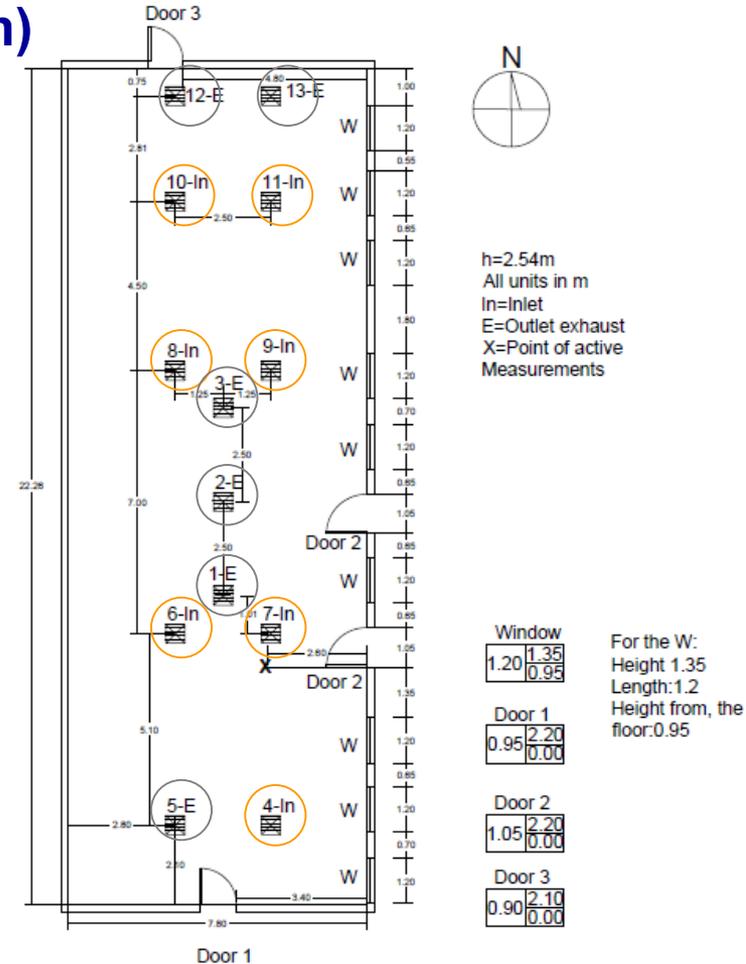
Presentation layout

- **The physical problem**
- **The simulation methodology**
- **Present the results of the 'basic case'**
 - Flow/ dispersion
 - Focus on spatial differences of concentrations
- **Additional cases in order to examine the influence of physical/modelling parameters:**
 - Alternative ventilation cases (different vent strength distribution/ geometry/ layout) – determination of best one
 - Alternative modelling cases (existence of desks/people, grid resolution, thermal influence, inlet conditions, CFD methodology (RANS/LES)) – reliability issues discussed

Physical problem

- Large office (22.26m x 7.80m x 2.54m)
 - 9 windows / 4 doors
- Mechanical ventilation
 - 7 inlet / 6 outlet vents
- Inlet change rate: 3.5 changes per hour

Mechanical ventilation hours	Every day, 07:00 - 19:00	
Flow rate per vent (kg/s)	1	- 0.050
	2	-0.067
	3	-0.059
	4	0.127
	5	-0.036
	6	0.047
	7	0.075
	8	0.073
	9	0.077
	10	0.044
	11	0.081
	12	-0.048
	13	-0.035
Total in (kg/s)	0.524	
Total out (kg/s)	0.295	

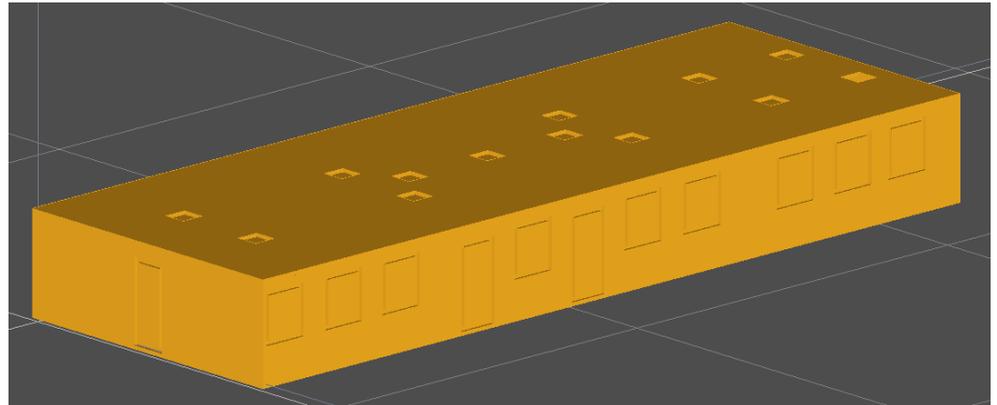


Pollutant Emission

- Pollutants such as PM, formaldehyde and other VOCs emitted mainly from floor and floor equipment (i.e. furnitures, desks etc)
- Assumption : uniform surface ground source
- Results will be presented non-dimensionalized with the same global average theoretical in-room concentration C_{av} that the office would have in case of full homogeneous commixture

The Simulation methodology

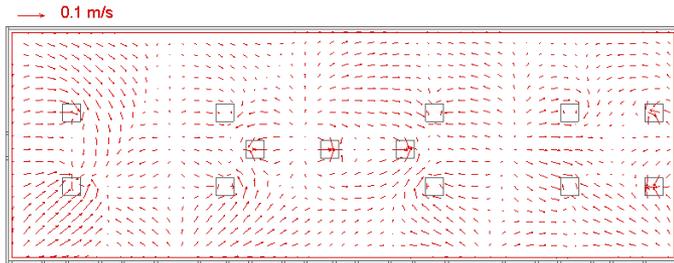
- Inlet/outlet flows *from vents* taken from experimental data
- Outflows *for doors/windows* simulated with COMIS
- 10cm gap around each door/window



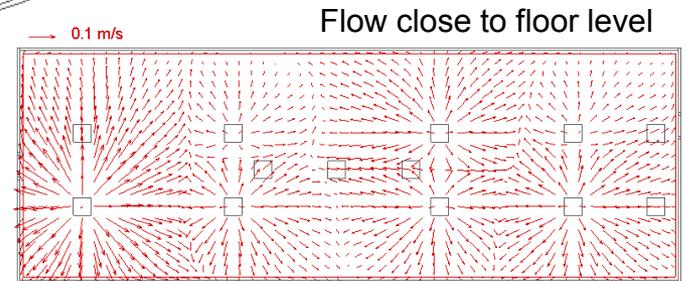
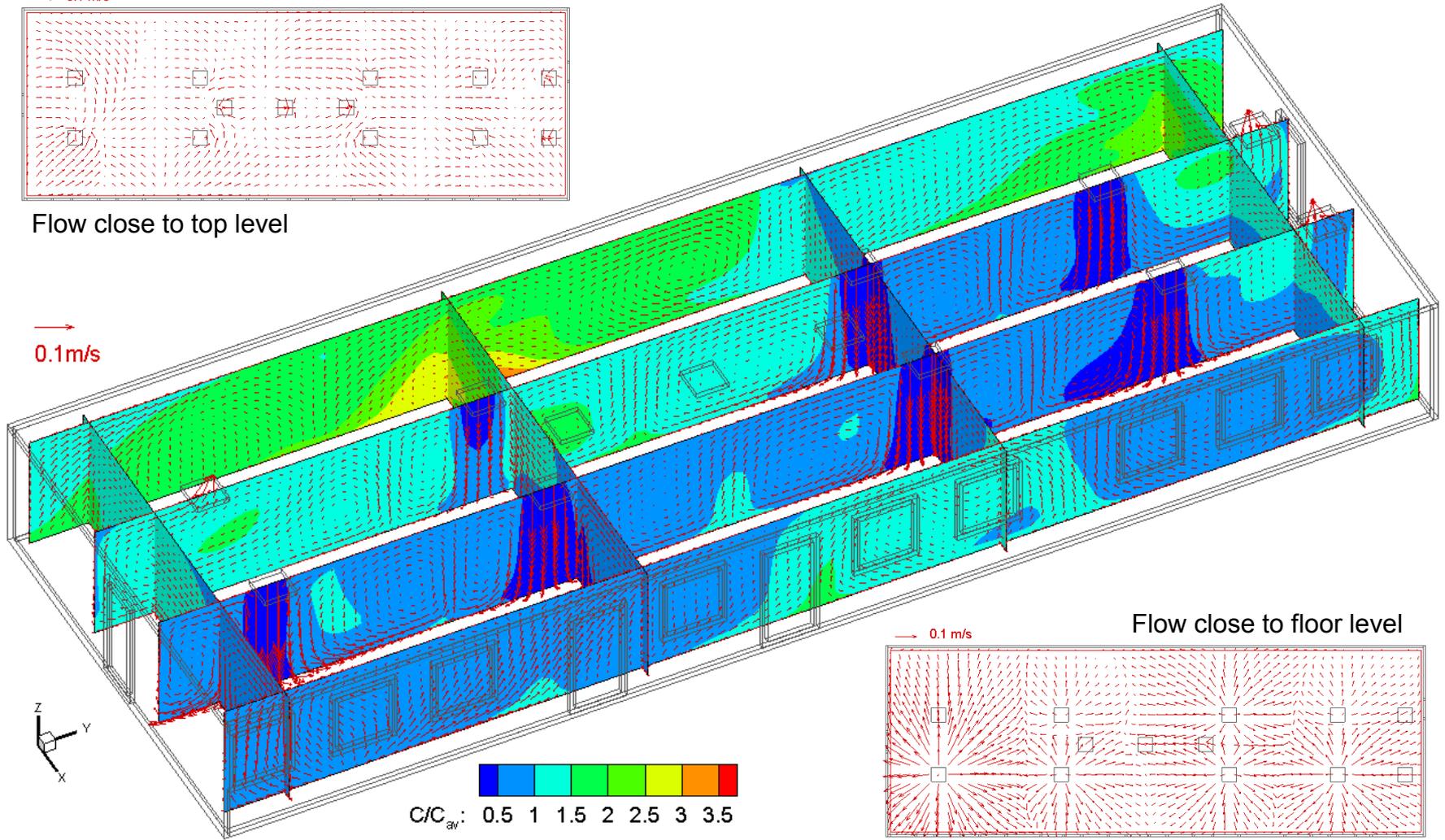
ADREA-HF

- Standard $k-\varepsilon$ (RANS)
- Basic case grid: 39 x 110 x 14 cells
- $z_0=0.001\text{m}$
- 12 'sensors' corresponding to the working occupants' positions

Basic case: Main results

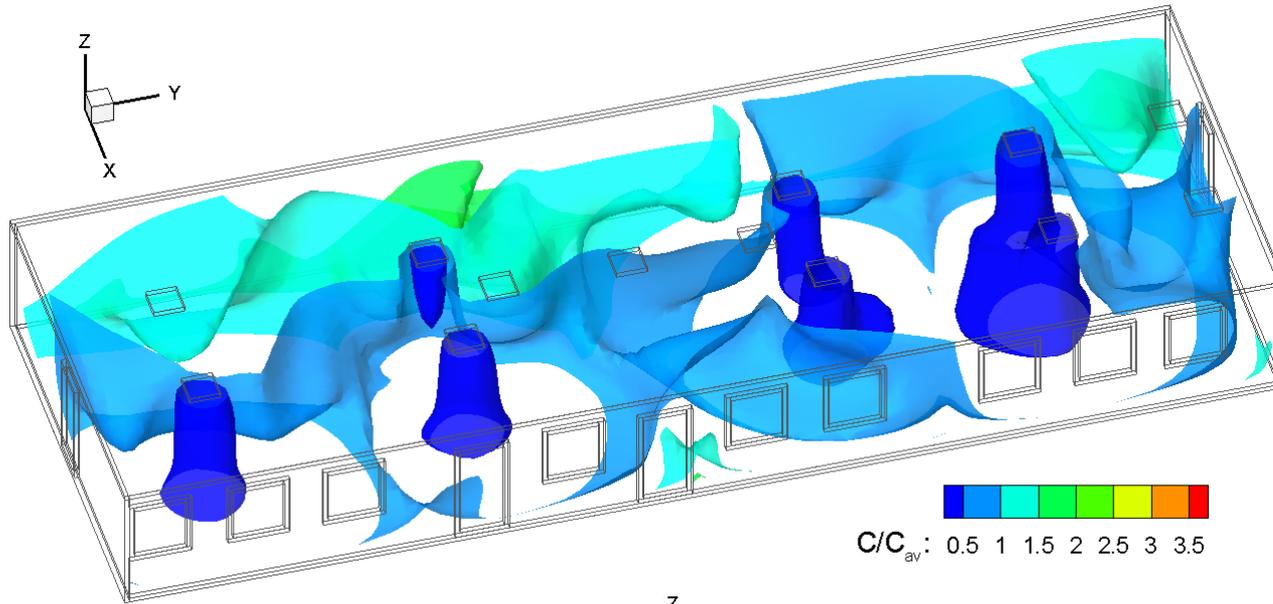


Flow close to top level

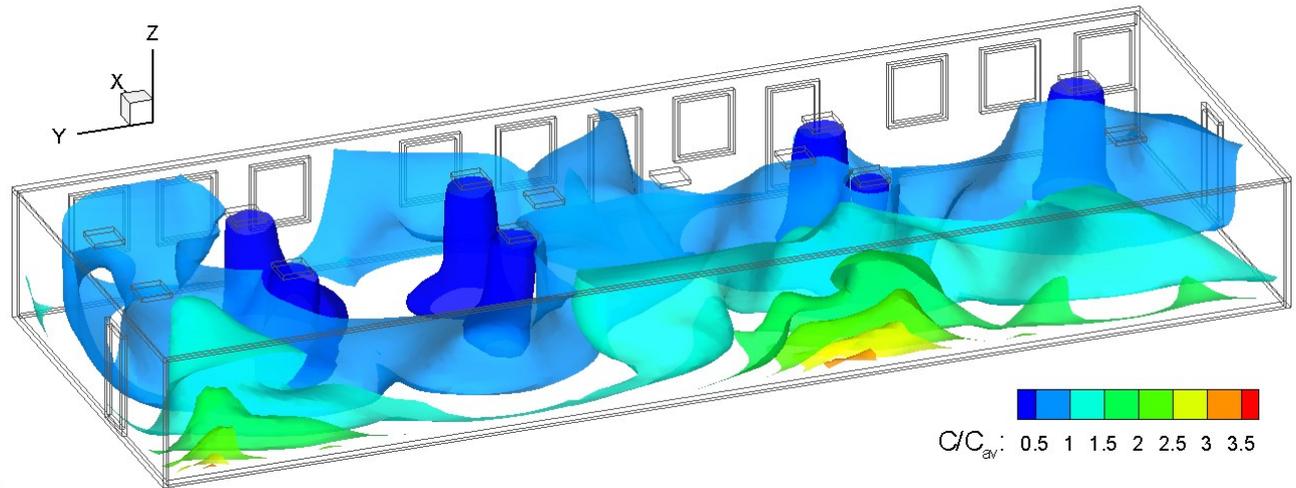


Flow close to floor level

Basic case: C/C_{av} isosurfaces

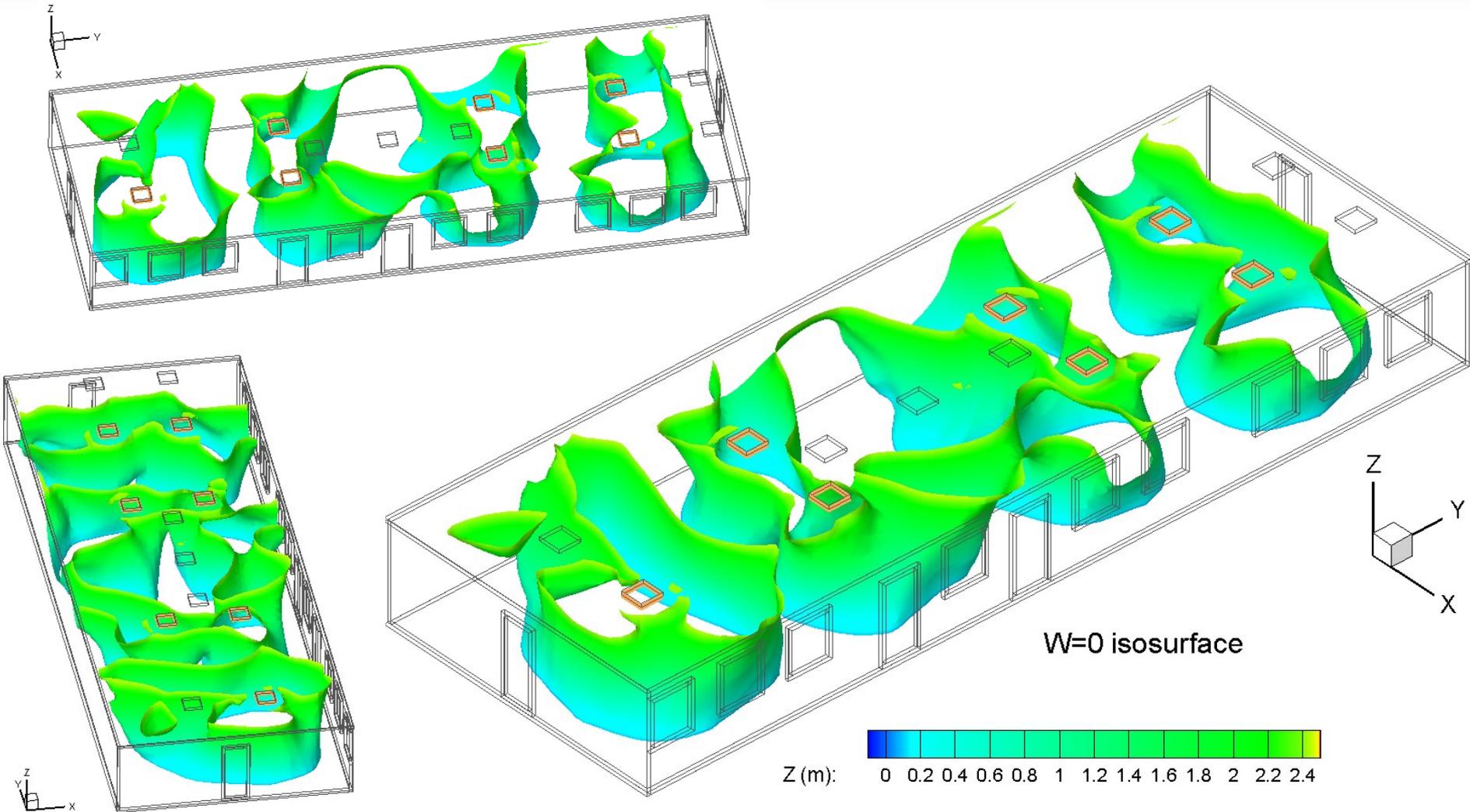


C/C_{av} : 0.5 1 1.5 2 2.5 3 3.5



C/C_{av} : 0.5 1 1.5 2 2.5 3 3.5

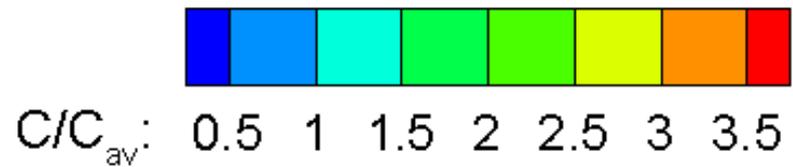
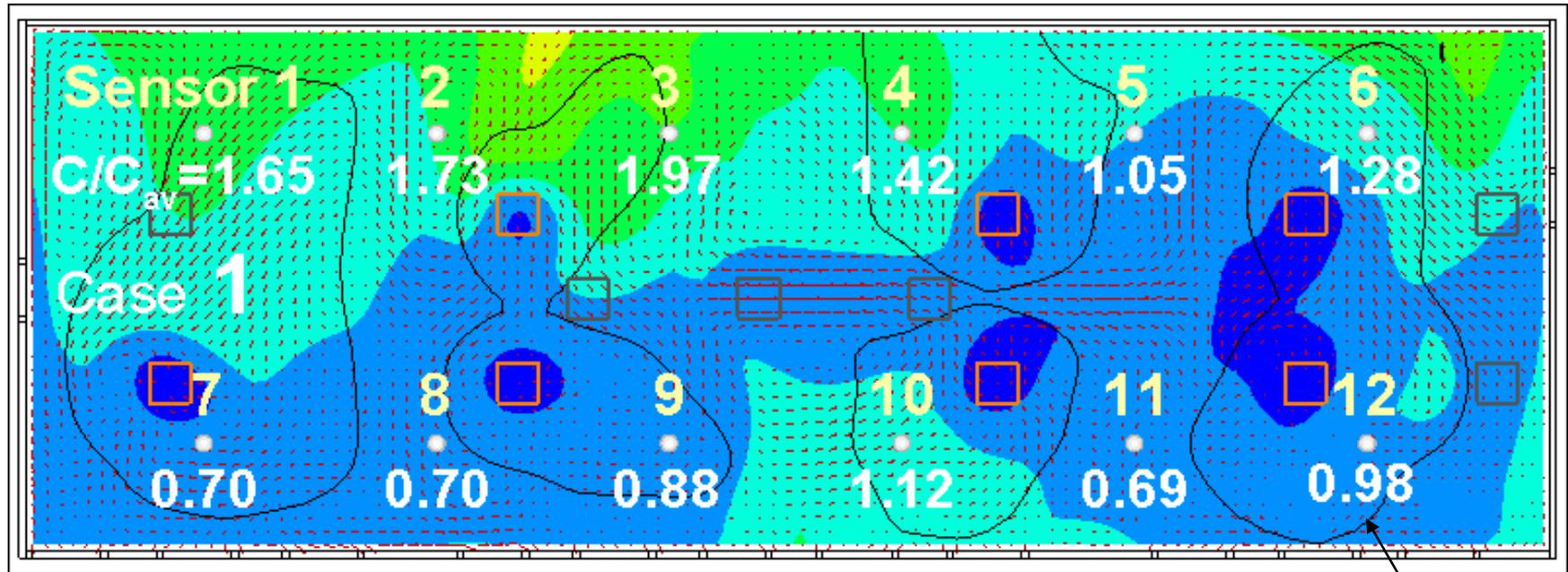
Basic case: $W=0$ isosurface



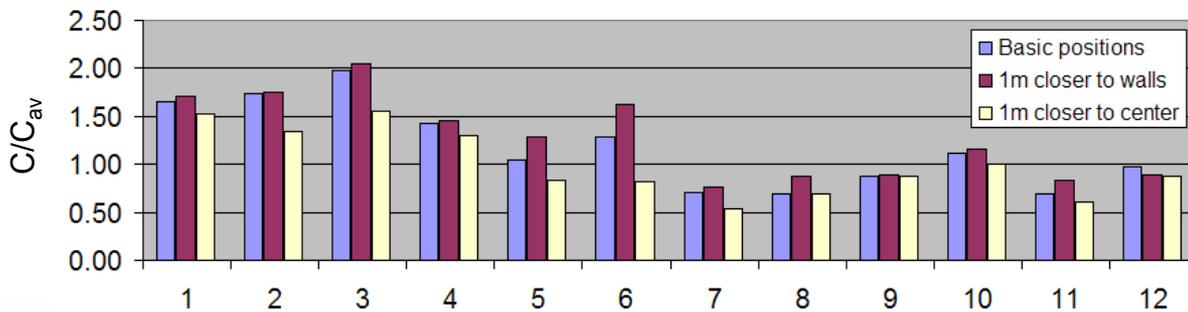
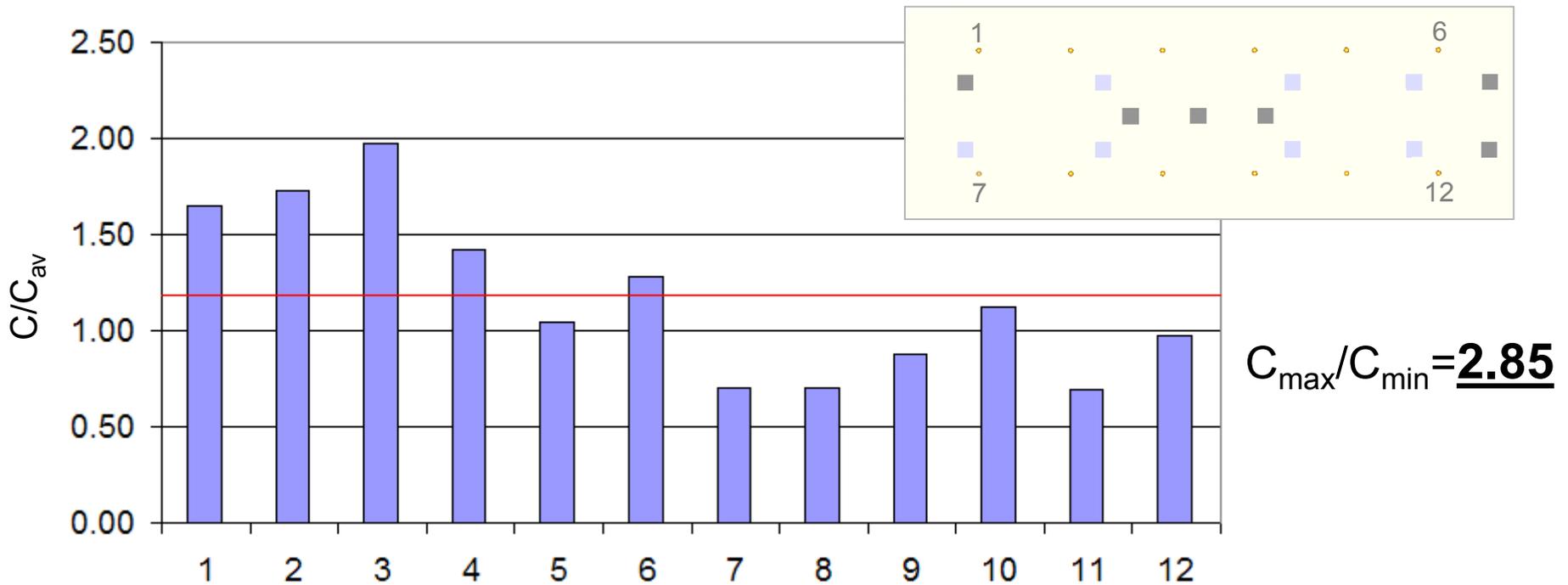
Basic case: Streamtraces



Basic case: Concentrations at Z = 1.1m



Basic case: Concentrations at sensors



If employees were:

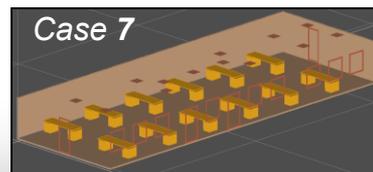
- 1m closer to walls: 8% higher av. C
- 1m closer to center: 15% lower av. C

All cases examined

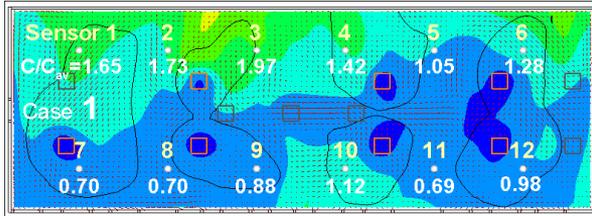
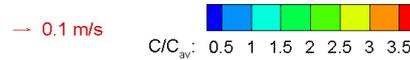
Alternative ventilation

- 1 – BASE
- 2 – Uniform inflow rate at vents
- 3 – Wall 2 vents 60cm closer to wall 2
- 4 – Other inlets/outlets (uniform inlets/ at wall 2 side)
- 5 – New vents design (inlet from the side walls)
- 6 – New vents design (inlet from center)
- 7 – Base case with 12 desks
- 8 – Base case with 12 desks and 12 “people”
- 9 – Fine grid (79 x 223 x 28 cells)
- 10 – Given T at inlets – energy equation also
- 11 – Given k at inlets
- 12 – Preliminary LES

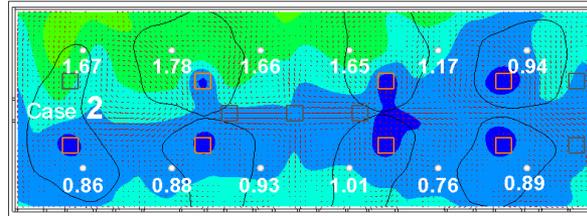
Alternative modelling



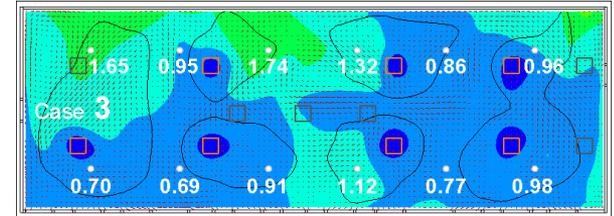
All cases: Concentrations at Z = 1.1m



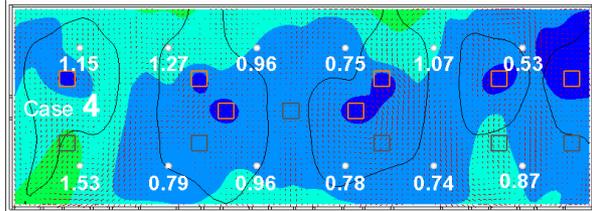
1- BASE (average: 1.18/ Max/min: 2.85)



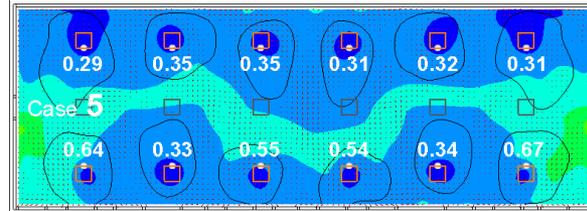
2- Uniform inflow rate at vents (1.18/2.34)



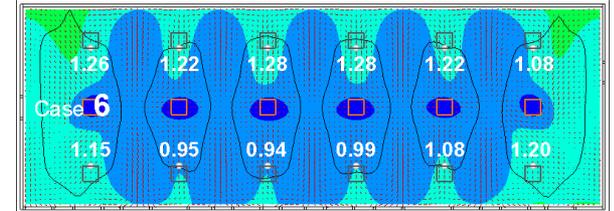
3- Wall 2 vents closer to wall (1.05/2.51)



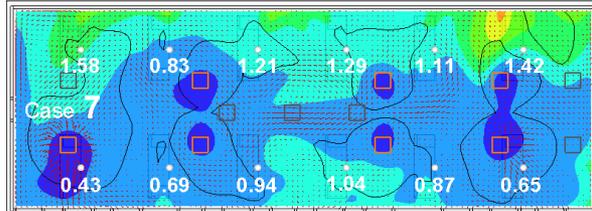
4- Other inlets/outlets (0.95/2.89)



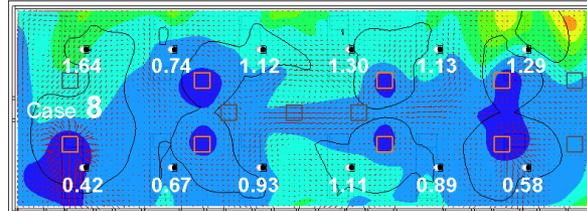
5- New vents (side inlets) (0.42/2.33)



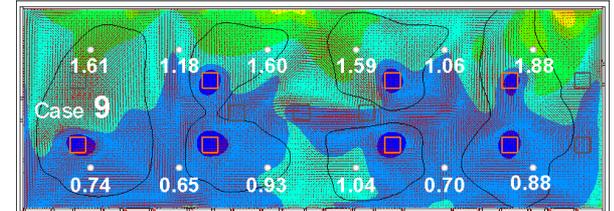
6- New vents (center inlets) (1.14/1.36)



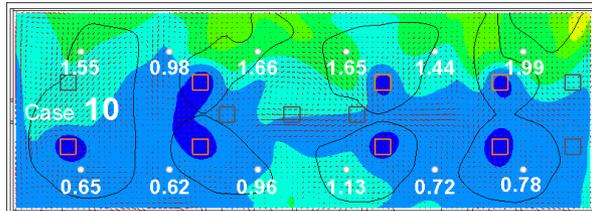
7- Base case with 12 desks (1.01/3.68)



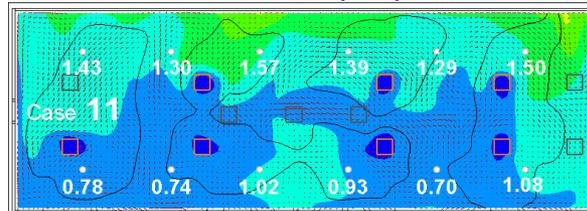
8- Base with desks & people (0.99/3.91)



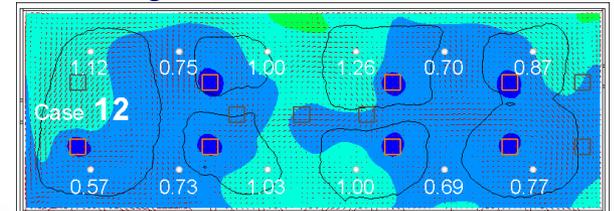
9- Fine grid (1.16/2.88)



10- Given T at inlets (1.18/3.20)



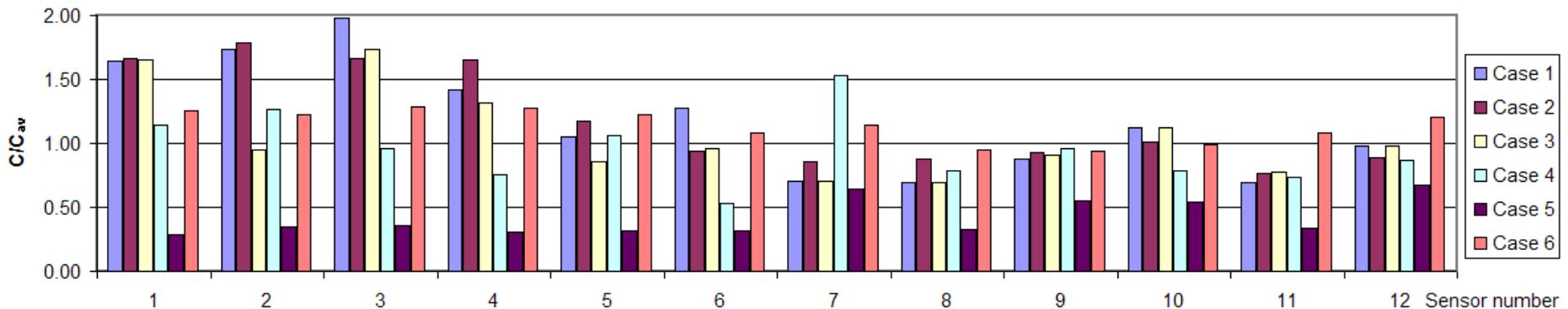
11- Given k at inlets (1.14/2.25)



12- Preliminary LES (0.87/2.23)

Alternative ventilation cases

Case	C/C _{av} at the 12 sensors:	Av.	min	max	max/ min
1 – BASE		1.18	0.69	1.97	2.85
2 – Uniform inflow rate at vents		1.18	0.76	1.78	2.34
3 – Wall 2 vents 60cm closer to wall 2		1.05	0.69	1.74	2.51
4 – Other inlets/outlets (uniform inlets/ at wall 2 side)		0.95	0.53	1.53	2.89
5 – New vents design (inlet from the side walls)		0.42	0.29	0.67	2.33
6 – New vents design (inlet from center)		1.14	0.94	1.28	1.36

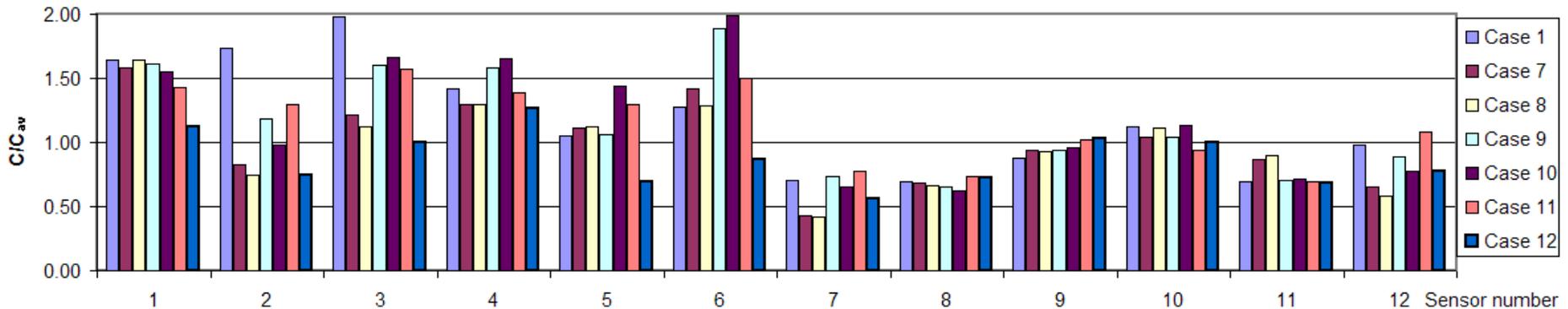


Alternative ventilation: comments

- **CFD very valuable tool for alternative scenarios**
- **More uniform ventilation increases uniformity at C**
- **Spotting problematic areas drives the thoughts for improvements**
- **Small improvement, of the order of 10-20 % can be achieved with small interventions**
- **A complete redesign of the ventilation system in this case results in 3 times lower C at working positions (best case from those examined)**

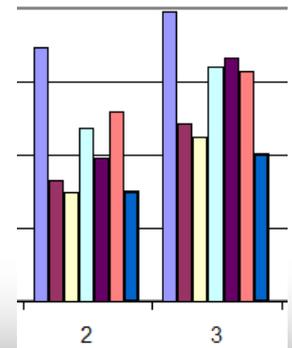
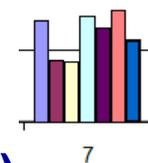
Alternative modelling cases

Case	C/C_{av} at the 12 sensors:	Av.	min	max	max/min
1 – BASE		1.18	0.69	1.97	2.85
7 – Base case with 12 desks		1.01	0.43	1.58	3.68
8 – Base case with 12 desks and 12 “people”		0.99	0.42	1.64	3.91
9 – Fine grid		1.16	0.65	1.88	2.88
10 – Given T at inlets – energy equation also		1.18	0.62	1.99	3.20
11 – Given k at inlets		1.14	0.70	1.57	2.25
12 – Preliminary LES		0.87	0.57	1.26	1.36



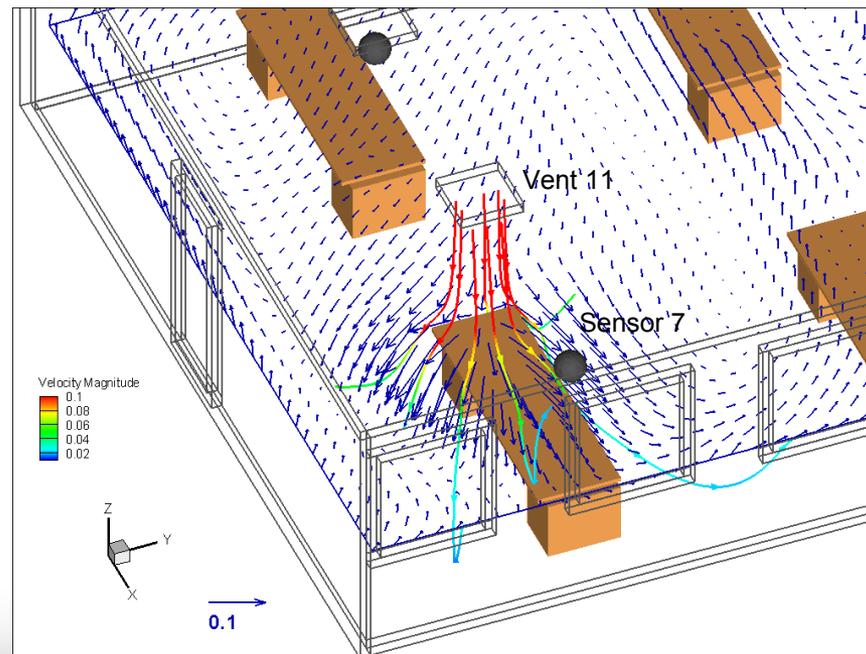
Alternative modelling: comments

- **General flow features are retained...**
- **...but differences are also present**
 - especially at sensor 2 (and 3), but also 6 (and 7)
- **More geometrical details → lower uniformity at C**
- **Temperature/ inlet conditions more critical than grid**
- **Preliminary LES has in general lower av. C in room**
- **Differences seem to relate more to the position than to the choice of the modelling parameters**
 - Next slides, focus on:
 - sensor 7 (lower C if desks are considered)
 - sensors 2,3 (higher differences among runs)



Alternative modelling: Focus on sensor 7 values

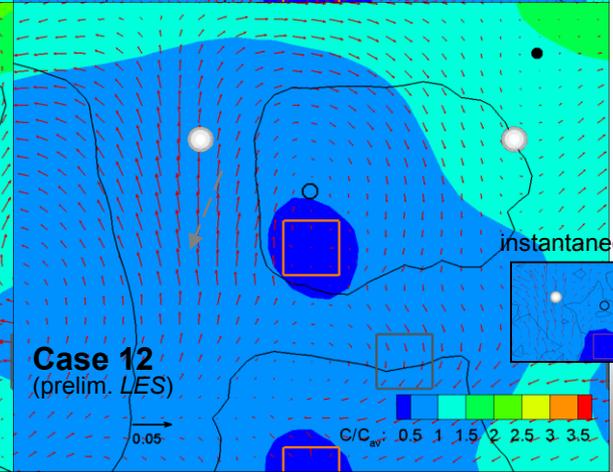
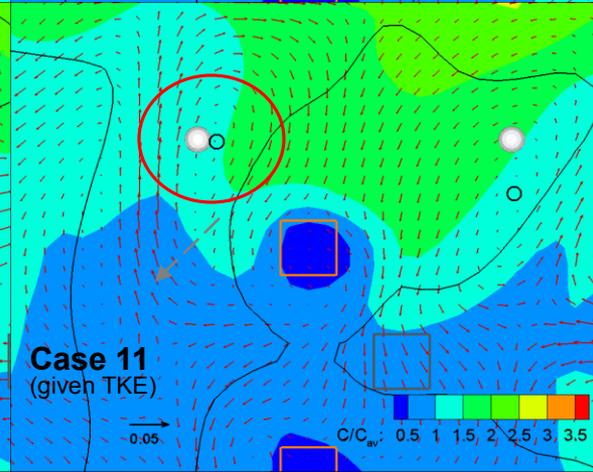
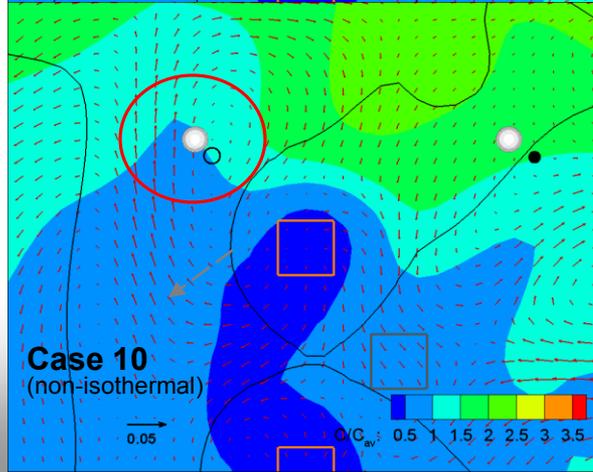
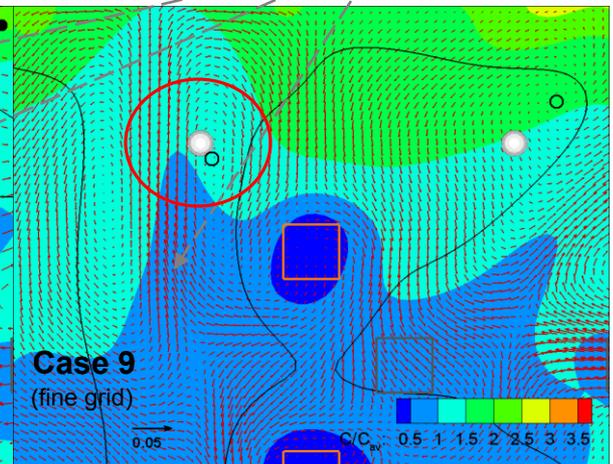
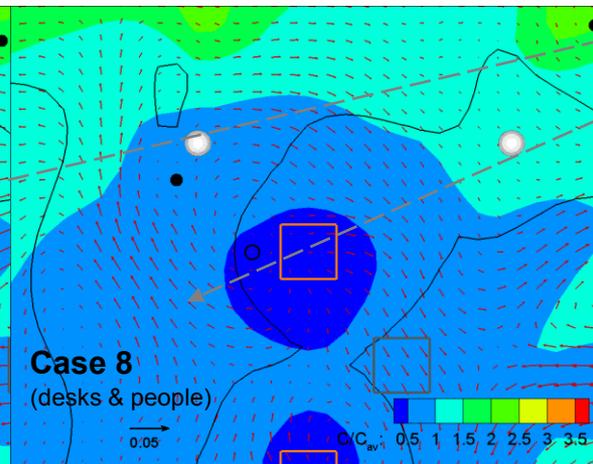
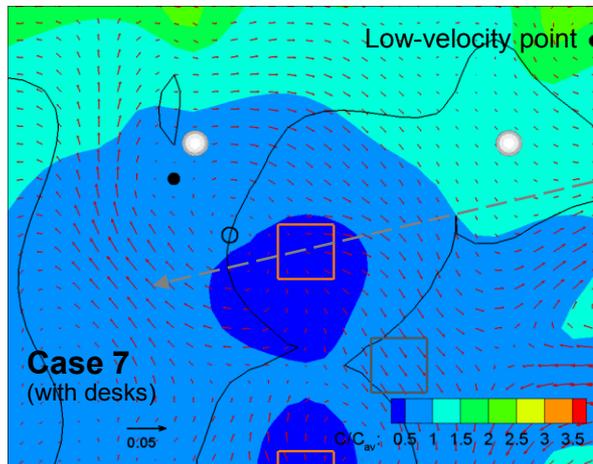
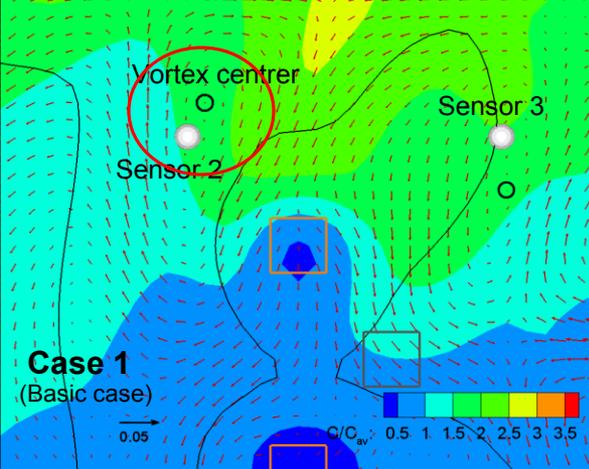
- **Sensor 7 presents lower concentrations (about 40%) when desks are incorporated in the model**
 - The desk that corresponds to sensor 7 is just below the vent number 11; thus the fresh air spreads above the desk and keeps the C values of sensor 7 very low



Alternative modelling: Focus on sensors 2 and 3

Differences among runs, esp. for sensor 2

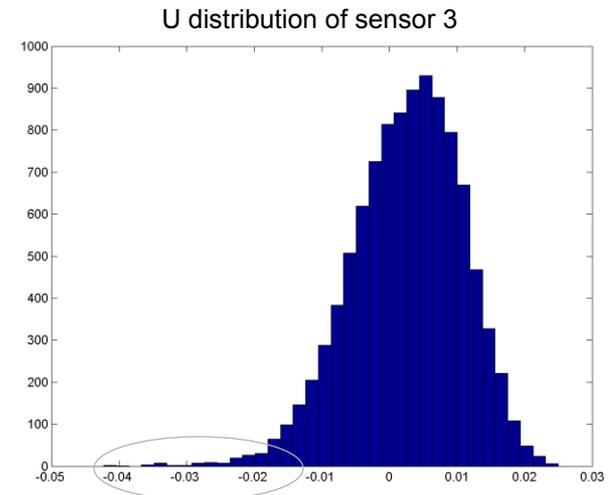
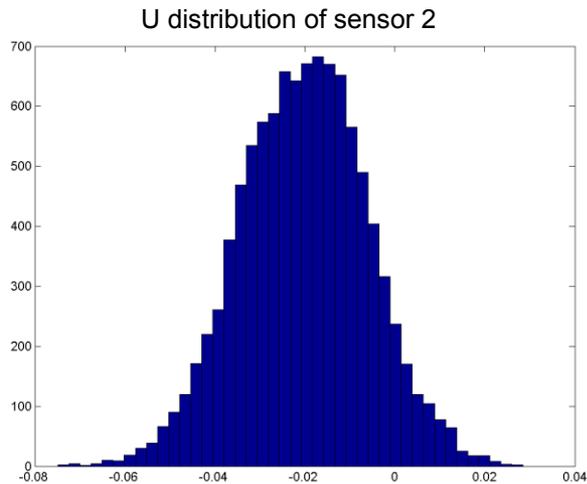
- Unsteady flow – close to (unsteady) vortex center
- Sensor 2: clean air transferred from elsewhere



From LES case 12:

U distributions at sensors 2, 3

- Actually, flow at area of sensor 2 (& 3) is unsteady
- From LES, U probability density functions from sensors 2 and 3 are the less Gaussian



Conclusions

- **Exposure at large offices presents high heterogeneity**
 - In this case: max/ min is 4, if in-room geometry is considered
- **CFD is a very valuable tool**
 - Analyze the flow, determine best working positions
 - Propose alternative ventilation and even new designs
- **Influential physical/ simulation parameters:**
 - Geometry/ layout/ strength of vents
 - In-room detailed geometry
 - Thermal effects
 - RANS vs. LES
- **Unsteadiness of flow causes CFD reliability issues**
- **LES should be further examined**
- **There is a need for a validation database**

Acknowledgements

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**OFFICAIR (On the reduction of health effects
from combined exposure to indoor air
pollutants in modern office buildings)**

Funded under the topic :

**ENV.2010.1.2.2-1: Indoor air pollution and health risks
of modern office buildings**

Ευχαριστώ
Thank you
